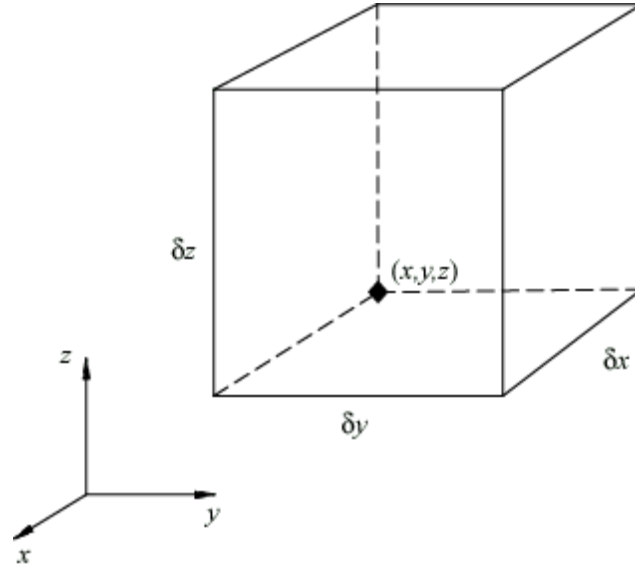


**Problem 1.1**



Inflow through  $x = \text{constant}$ :

$$\rho u \delta y \delta z$$

Outflow through  $x + \delta x = \text{constant}$ :

$$\rho u \delta y \delta z + \frac{\partial}{\partial x}(\rho u \delta y \delta z) \delta x + \dots$$

Net inflow through  $x = \text{constant}$  surfaces:

$$-\frac{\partial}{\partial x}(\rho u) \delta x \delta y \delta z + \dots$$

Net inflow through  $y = \text{constant}$  surfaces:

$$-\frac{\partial}{\partial y}(\rho v) \delta x \delta y \delta z + \dots$$

Net inflow through  $z = \text{constant}$  surfaces:

$$-\frac{\partial}{\partial z}(\rho w) \delta x \delta y \delta z + \dots$$

But the rate at which the mass is accumulating inside the control volume is:

$$\frac{\partial}{\partial t}(\rho \delta x \delta y \delta z)$$

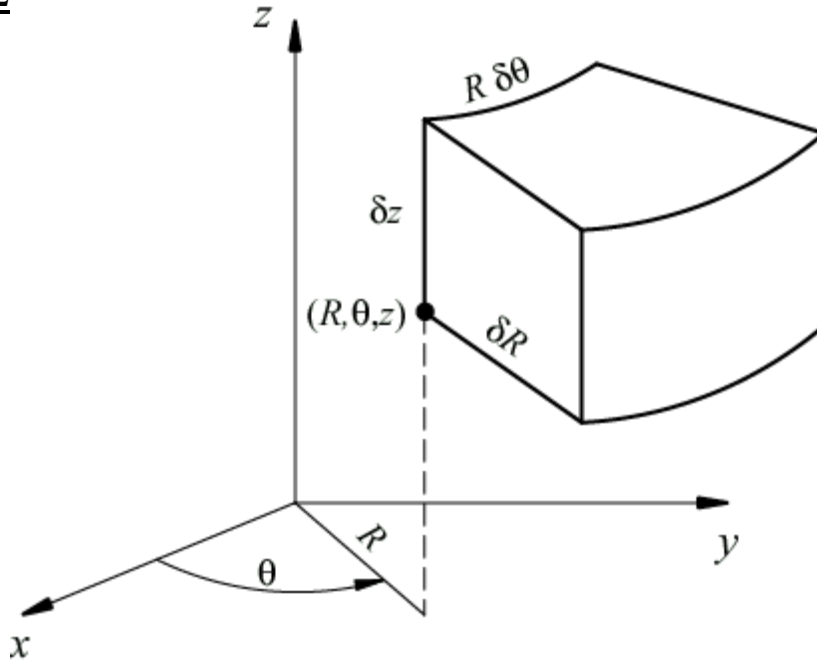
Then the equation of mass conservation becomes:

$$\frac{\partial \rho}{\partial t} \delta x \delta y \delta z = - \left[ \frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) + \frac{\partial}{\partial z}(\rho w) \right] \delta x \delta y \delta z + \dots$$

Taking the limits as the quantities  $\delta x$ ,  $\delta y$  and  $\delta z$  become vanishingly small, we get:

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) + \frac{\partial}{\partial z}(\rho w) = 0$$

**Problem 1.2**



Inflow through  $R = \text{constant}$ :  $\rho u_R R \delta\theta \delta z$

Outflow through  $R + \delta R = \text{constant}$ :  $\rho u_R R \delta\theta \delta z + \frac{\partial}{\partial R}(\rho u_R R \delta\theta \delta z) \delta R + \dots$

Net inflow through  $R = \text{constant}$  surfaces:  $-\frac{\partial}{\partial R}(\rho R u_R) \delta R \delta\theta \delta z + \dots$

Net inflow through  $\theta = \text{constant}$  surfaces:  $-\frac{\partial}{\partial \theta}(\rho u_\theta) \delta R \delta\theta \delta z + \dots$

Net inflow through  $z = \text{constant}$  surfaces:  $-\frac{\partial}{\partial z}(\rho u_z) R \delta R \delta\theta \delta z + \dots$

But the rate at which the mass is accumulating inside the control volume is:

$$\frac{\partial}{\partial t}(\rho R \delta R \delta\theta \delta z)$$

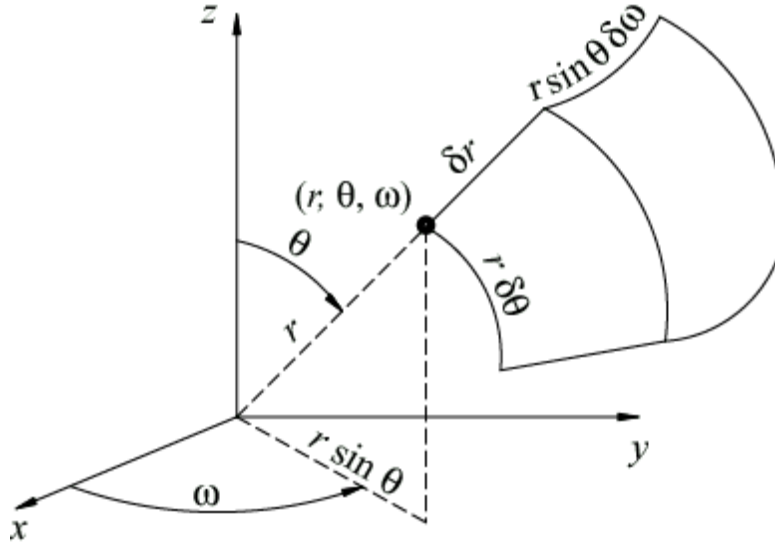
Then the equation of mass conservation becomes:

$$\frac{\partial \rho}{\partial t} R \delta R \delta\theta \delta z = - \left[ \frac{\partial}{\partial R}(\rho R u_R) + \frac{\partial}{\partial \theta}(\rho u_\theta) + R \frac{\partial}{\partial z}(\rho u_z) \right] \delta R \delta\theta \delta z + \dots$$

Taking the limits as the quantities  $\delta R$ ,  $\delta\theta$  and  $\delta z$  become vanishingly small, we get:

$$\boxed{\frac{\partial \rho}{\partial t} + \frac{1}{R} \frac{\partial}{\partial R}(\rho R u_R) + \frac{1}{R} \frac{\partial}{\partial \theta}(\rho u_\theta) + \frac{\partial}{\partial z}(\rho u_z) = 0}$$

**Problem 1.3**



Inflow through  $r = \text{constant}$ :

$$\rho u_r r^2 \sin \theta \delta \theta \delta \omega$$

Outflow through  $r + \delta r = \text{constant}$ :

$$\rho u_r r^2 \sin \theta \delta \theta \delta \omega$$

$$+ \frac{\partial}{\partial r} (\rho r^2 u_r \sin \theta \delta \theta \delta \omega) \delta r + \dots$$

Net inflow through  $r = \text{constant}$  surfaces:  $-\frac{\partial}{\partial r} (\rho r^2 u_r) \sin \theta \delta r \delta \theta \delta \omega + \dots$

Net inflow through  $\theta = \text{constant}$  surfaces:  $-\frac{\partial}{\partial \theta} (\rho u_\theta \sin \theta) r \delta r \delta \theta \delta \omega + \dots$

Net inflow through  $\omega = \text{constant}$  surfaces:  $-\frac{\partial}{\partial \omega} (\rho u_\omega) r \delta r \delta \theta \delta \omega + \dots$

But the rate at which the mass is accumulating inside the control volume is:

$$\frac{\partial}{\partial t} (\rho r^2 \sin \theta \delta r \delta \theta \delta \omega)$$

Then the equation of mass conservation becomes:

$$\frac{\partial \rho}{\partial t} r^2 \sin \theta \delta r \delta \theta \delta \omega = - \left[ \frac{\partial}{\partial r} (\rho r^2 u_r) \sin \theta + r \frac{\partial}{\partial \theta} (\rho u_\theta \sin \theta) + r \frac{\partial}{\partial \omega} (\rho u_\omega) \right] \delta r \delta \theta \delta \omega + \dots$$

Taking the limits as the quantities  $\delta r$ ,  $\delta \theta$  and  $\delta \omega$  become vanishingly small, we get:

$$\frac{\partial \rho}{\partial t} + \frac{1}{r^2} \frac{\partial}{\partial r} (\rho r^2 u_r) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \theta} (\rho u_\theta \sin \theta) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \omega} (\rho u_\omega) = 0$$

**Problem 1.4**

Using the given transformation equations gives:

$$R^2 = x^2 + y^2 \quad \text{and} \quad \tan \theta = \frac{y}{x}$$

$$\therefore 2R \frac{\partial R}{\partial x} = 2x = 2R \cos \theta \quad \Rightarrow \quad \frac{\partial R}{\partial x} = \cos \theta$$

$$\text{and } \sec^2 \theta \frac{\partial \theta}{\partial x} = -\frac{y}{x^2} = -\frac{1}{R} \frac{\sin \theta}{\cos^2 \theta} \quad \Rightarrow \quad \frac{\partial \theta}{\partial x} = -\frac{1}{R} \sin \theta$$

Using these results, the derivatives with respect to  $x$  and  $y$  transform as follows:

$$\frac{\partial}{\partial x} = \frac{\partial R}{\partial x} \frac{\partial}{\partial R} + \frac{\partial \theta}{\partial x} \frac{\partial}{\partial \theta} = \cos \theta \frac{\partial}{\partial R} - \frac{\sin \theta}{R} \frac{\partial}{\partial \theta}$$

$$\frac{\partial}{\partial y} = \frac{\partial R}{\partial y} \frac{\partial}{\partial R} + \frac{\partial \theta}{\partial y} \frac{\partial}{\partial \theta} = \sin \theta \frac{\partial}{\partial R} + \frac{\cos \theta}{R} \frac{\partial}{\partial \theta}$$

$$\frac{\partial}{\partial z} = \frac{\partial}{\partial z}$$

Using these results and the relationships between the Cartesian and cylindrical vector components, we get the following expressions for the Cartesian terms in the continuity equation:

$$\frac{\partial}{\partial x}(\rho u) = \cos \theta \frac{\partial}{\partial R}[\rho(u_R \cos \theta - u_\theta \sin \theta)] - \frac{\sin \theta}{R} \frac{\partial}{\partial \theta}[(\rho(u_R \cos \theta - u_\theta \sin \theta))]$$

$$\frac{\partial}{\partial y}(\rho v) = \sin \theta \frac{\partial}{\partial R}[\rho(u_R \sin \theta + u_\theta \cos \theta)] + \frac{\cos \theta}{R} \frac{\partial}{\partial \theta}[(\rho(u_R \sin \theta + u_\theta \cos \theta))]$$

Adding these two terms and simplifying produces the following equation:

$$\frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) = \frac{\partial}{\partial R}(\rho u_R) + \frac{\rho u_R}{R} + \frac{1}{R} \frac{\partial}{\partial \theta}(\rho u_\theta)$$

$$= \frac{1}{R} \frac{\partial}{\partial R}(\rho R u_R) + \frac{1}{R} \frac{\partial}{\partial \theta}(\rho u_\theta)$$

Substituting this result into the full continuity equation yields the following result:

$$\frac{\partial \rho}{\partial t} + \frac{1}{R} \frac{\partial}{\partial R}(\rho R u_R) + \frac{1}{R} \frac{\partial}{\partial \theta}(\rho u_\theta) + \frac{\partial}{\partial z}(\rho u_z) = 0$$

**Problem 1.5**

The equations that connect the two coordinate systems are as follows:

$$\begin{aligned} x &= r \sin \theta \cos \omega & y &= r \sin \theta \sin \omega & z &= r \cos \theta \\ r^2 &= x^2 + y^2 + z^2 & \tan \omega &= \frac{y}{x} & \cos^2 \theta &= \frac{z^2}{(x^2 + y^2 + z^2)} \end{aligned}$$

Using the relations given above, the following identities are obtained for the various partial derivatives:

$$\begin{aligned} \frac{\partial r}{\partial x} &= \sin \theta \cos \omega & \frac{\partial r}{\partial y} &= \sin \theta \sin \omega & \frac{\partial r}{\partial z} &= \cos \theta \\ \frac{\partial \theta}{\partial x} &= -\frac{1}{r} \cos \theta \cos \omega & \frac{\partial \theta}{\partial y} &= -\frac{1}{r} \cos \theta \sin \omega & \frac{\partial \theta}{\partial z} &= -\frac{1}{r} \sin \theta \\ \frac{\partial \omega}{\partial x} &= \frac{1 \sin \omega}{r \sin \theta} & \frac{\partial \omega}{\partial y} &= \frac{1 \cos \omega}{r \cos \theta} & \frac{\partial \omega}{\partial z} &= 0 \end{aligned}$$

Thus the following expressions are obtained for the various Cartesian derivatives:

$$\begin{aligned} \frac{\partial}{\partial x} &= \frac{\partial r}{\partial x} \frac{\partial}{\partial r} + \frac{\partial \theta}{\partial x} \frac{\partial}{\partial \theta} + \frac{\partial \omega}{\partial x} \frac{\partial}{\partial \omega} \\ &= \sin \theta \cos \omega \frac{\partial}{\partial r} - r \cos \theta \cos \omega \frac{\partial}{\partial \omega} + \frac{1 \sin \omega}{r \sin \theta} \frac{\partial}{\partial \omega} \\ \frac{\partial}{\partial y} &= \frac{\partial r}{\partial y} \frac{\partial}{\partial r} + \frac{\partial \theta}{\partial y} \frac{\partial}{\partial \theta} + \frac{\partial \omega}{\partial y} \frac{\partial}{\partial \omega} \\ &= \sin \theta \sin \omega \frac{\partial}{\partial r} - \frac{1}{r} \cos \theta \sin \omega \frac{\partial}{\partial \theta} + \frac{1 \cos \omega}{r \sin \theta} \frac{\partial}{\partial \omega} \\ \frac{\partial}{\partial z} &= \frac{\partial r}{\partial z} \frac{\partial}{\partial r} + \frac{\partial \theta}{\partial z} \frac{\partial}{\partial \theta} + \frac{\partial \omega}{\partial z} \frac{\partial}{\partial \omega} \\ &= \cos \theta \frac{\partial}{\partial r} - \frac{1}{r} \sin \theta \frac{\partial}{\partial \theta} \end{aligned}$$

Next we need the Cartesian velocity components expressed in terms of spherical components. This may be achieved by noting that the velocity vector may be written as follows:

$$\mathbf{u} = u \mathbf{e}_x + v \mathbf{e}_y + w \mathbf{e}_z = u_r \mathbf{e}_r + u_\theta \mathbf{e}_\theta + u_\omega \mathbf{e}_\omega$$

Then, if we express the base vectors in spherical coordinates in terms of the base vectors in Cartesian coordinates, equating components in the previous equation will yield the required relationships. Thus, noting that:

## BASIC CONSERVATION LAWS

$$\mathbf{r} = x\mathbf{e}_x + y\mathbf{e}_y + z\mathbf{e}_z = r \sin \theta \cos \omega \mathbf{e}_x + r \sin \theta \sin \omega \mathbf{e}_y + r \cos \theta \mathbf{e}_z$$

Also, recalling from Appendix A that:

$$\mathbf{e}_i = \frac{\partial \mathbf{r}}{\partial x_i} \bigg/ \left| \frac{\partial \mathbf{r}}{\partial x_i} \right| \quad \text{it follows that}$$

$$\mathbf{e}_r = \sin \theta \cos \omega \mathbf{e}_x + \sin \theta \sin \omega \mathbf{e}_y + \cos \theta \mathbf{e}_z$$

$$\mathbf{e}_\theta = \cos \theta \cos \omega \mathbf{e}_x + \cos \theta \sin \omega \mathbf{e}_y - \sin \theta \mathbf{e}_z$$

$$\mathbf{e}_\omega = -\sin \omega \mathbf{e}_x + \cos \omega \mathbf{e}_y$$

Substituting these expressions into the equation obtained above for the velocity vector, and equating coefficients of like base vectors, yields the following relationships connecting the Cartesian and spherical velocity components:

$$u = u_r \sin \theta \cos \omega + u_\theta \cos \theta \cos \omega - u_\omega \sin \omega$$

$$v = u_r \sin \theta \sin \omega + u_\theta \cos \theta \sin \omega + u_\omega \cos \omega$$

$$w = u_r \cos \theta - u_\theta \sin \theta$$

Using these results, and those obtained for the Cartesian derivatives, produces the following expression for the Cartesian terms that appear in the continuity equation:

$$\frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) + \frac{\partial}{\partial z}(\rho w) = \frac{1}{r^2} \frac{\partial}{\partial r}(\rho r^2 u_r) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \theta}(\rho u_\theta \sin \theta) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \omega}(\rho u_\omega)$$

Substituting this result into the full continuity equation yields the following expression:

$$\boxed{\frac{\partial \rho}{\partial t} + \frac{1}{r^2} \frac{\partial}{\partial r}(\rho r^2 u_r) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \theta}(\rho u_\theta \sin \theta) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \omega}(\rho u_\omega) = 0}$$

### **Problem 1.6**

From Appendix A, the following value is obtained for the convective derivative:

$$(\mathbf{a} \cdot \nabla) \mathbf{a} = \frac{1}{h_1} \left\{ \left( a_1 \frac{\partial a_1}{\partial x_1} + a_2 \frac{\partial a_2}{\partial x_1} + a_3 \frac{\partial a_3}{\partial x_1} \right) - \frac{a_2}{h_2} \left[ \frac{\partial}{\partial x_1} (h_2 a_2) - \frac{\partial}{\partial x_2} (h_1 a_1) \right] \right.$$

$$\left. + \frac{a_3}{h_3} \left[ \frac{\partial}{\partial x_3} (h_1 a_1) - \frac{\partial}{\partial x_1} (h_3 a_3) \right] \right\} \mathbf{e}_1 + (\dots) \mathbf{e}_2 + (\dots) \mathbf{e}_3$$

Applying this result to cylindrical coordinates, we interpret the various terms as follows:

$$\begin{array}{lll}
 a_1 = u_R & a_2 = u_\theta & a_3 = u_z \\
 x_1 = R & x_2 = \theta & x_3 = z \\
 h_1 = 1 & h_2 = R & h_3 = 1 \\
 \mathbf{e}_1 = \mathbf{e}_R & \mathbf{e}_2 = \mathbf{e}_\theta & \mathbf{e}_3 = \mathbf{e}_z
 \end{array}$$

Using these results, the required term becomes:

$$\mathbf{e}_R \cdot [(\mathbf{u} \cdot \nabla) \mathbf{u}] = \left( u_R \frac{\partial u_R}{\partial R} + u_\theta \frac{\partial u_\theta}{\partial R} + u_z \frac{\partial u_z}{\partial R} \right) - \frac{u_\theta}{R} \left[ \frac{\partial}{\partial R} (R u_\theta) - \frac{\partial u_R}{\partial \theta} \right] + u_z \left( \frac{\partial u_R}{\partial z} - \frac{\partial u_z}{\partial R} \right)$$

Simplifying the right side of this equation produces the required result:

$$\mathbf{e}_R \cdot [(\mathbf{u} \cdot \nabla) \mathbf{u}] = u_R \frac{\partial u_R}{\partial R} + \frac{u_\theta}{R} \frac{\partial u_R}{\partial \theta} - \frac{u_\theta^2}{R} + u_z \frac{\partial u_R}{\partial z}$$

### **Problem 1.7**

We use the same starting equation from Appendix A as in the previous problem. However, since we are dealing with spherical coordinates here, the various terms are as follows:

$$\begin{array}{lll}
 a_1 = u_r & a_2 = u_\theta & a_3 = u_\omega \\
 x_1 = r & x_2 = \theta & x_3 = \omega \\
 h_1 = 1 & h_2 = r & h_3 = r \sin \theta \\
 \mathbf{e}_1 = \mathbf{e}_r & \mathbf{e}_2 = \mathbf{e}_\theta & \mathbf{e}_3 = \mathbf{e}_\omega
 \end{array}$$

Using these results, the required term becomes:

$$\mathbf{e}_r \cdot [(\mathbf{u} \cdot \nabla) \mathbf{u}] = \left( u_r \frac{\partial u_r}{\partial r} + u_\theta \frac{\partial u_\theta}{\partial r} + u_\omega \frac{\partial u_\omega}{\partial r} \right) - \frac{u_\theta}{r} \left[ \frac{\partial}{\partial r} (r u_\theta) - \frac{\partial u_r}{\partial \theta} \right] + \frac{u_\omega}{r \sin \theta} \left( \frac{\partial u_r}{\partial \omega} - \frac{\partial}{\partial r} (r u_\omega \sin \theta) \right)$$

Simplifying the right side of this equation yields the required result:

$$\mathbf{e}_r \cdot [(\mathbf{u} \cdot \nabla) \mathbf{u}] = u_r \frac{\partial u_r}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_r}{\partial \theta} - \frac{u_\theta^2 + u_\omega^2}{r} + \frac{u_\omega}{r \sin \theta} \frac{\partial u_r}{\partial \omega}$$

**Problem 1.8**

For a Newtonian fluid, the shear stress tensor is defined by the following equation:

$$\tau_{ij} = \lambda \delta_{ij} \frac{\partial u_k}{\partial x_k} + \mu \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

Evaluating the various terms in this expression for Cartesian coordinates  $(x, y, z)$  and Cartesian velocity components  $(u, v, w)$  yields the following results:

$$\begin{aligned} \tau_{xx} &= \lambda \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right) + 2\mu \frac{\partial u}{\partial x} \\ \tau_{yy} &= \lambda \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right) + 2\mu \frac{\partial v}{\partial y} \\ \tau_{zz} &= \lambda \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right) + 2\mu \frac{\partial w}{\partial z} \\ \tau_{xy} &= \tau_{yx} = \mu \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \\ \tau_{xz} &= \tau_{zx} = \mu \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right) \\ \tau_{yz} &= \tau_{zy} = \mu \left( \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right) \end{aligned}$$

For a monotonic gas, the Stokes relation requires that  $\lambda = -2\mu/3$ . Then the relations obtained above assume the following special form:

$$\begin{aligned} \tau_{xx} &= \frac{\mu}{3} \left( 4 \frac{\partial u}{\partial x} - 2 \frac{\partial v}{\partial y} - 2 \frac{\partial w}{\partial z} \right) \\ \tau_{yy} &= \frac{\mu}{3} \left( 4 \frac{\partial v}{\partial y} - 2 \frac{\partial u}{\partial x} - 2 \frac{\partial w}{\partial z} \right) \\ \tau_{zz} &= \frac{\mu}{3} \left( 4 \frac{\partial w}{\partial z} - 2 \frac{\partial u}{\partial x} - 2 \frac{\partial v}{\partial y} \right) \\ \tau_{xy} &= \tau_{yx} = \mu \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \\ \tau_{xz} &= \tau_{zx} = \mu \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right) \\ \tau_{yz} &= \tau_{zy} = \mu \left( \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right) \end{aligned}$$

**Problem 1.9**

For a Newtonian fluid, the dissipation function is defined by the following equation:

$$\Phi = \lambda \left( \frac{\partial u_k}{\partial x_k} \right)^2 + \mu \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \frac{\partial u_j}{\partial x_i}$$

Evaluating the various terms in this equation for the Cartesian coordinates  $(x, y, z)$  and the Cartesian velocity components  $(u, v, w)$ , yields the following value for  $\Phi$ :

$$\begin{aligned} \Phi = & \lambda \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right)^2 + 2\mu \left[ \left( \frac{\partial u}{\partial x} \right)^2 + \left( \frac{\partial v}{\partial y} \right)^2 + \left( \frac{\partial w}{\partial z} \right)^2 \right] \\ & + \mu \left[ \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right)^2 + \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right)^2 + \left( \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right)^2 \right] \end{aligned}$$

For a monotonic gas, the Stokes relation requires that  $\lambda = -2\mu/3$ . Then the general expression for  $\Phi$  obtained above assumes the following special form:

$$\begin{aligned} \Phi = & \mu \left\{ -\frac{2}{3} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right)^2 + 2 \left( \frac{\partial u}{\partial x} \right)^2 + 2 \left( \frac{\partial v}{\partial y} \right)^2 + 2 \left( \frac{\partial w}{\partial z} \right)^2 \right. \\ & \left. + \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right)^2 + \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right)^2 + \left( \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right)^2 \right\} \end{aligned}$$

**Problem 1.10**

For steady flow of an inviscid and incompressible fluid, but one for which the density is not constant, the two-dimensional governing equations are:

$$\frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) = 0$$

$$\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} = -\frac{\partial p}{\partial x}$$

$$\rho u \frac{\partial v}{\partial x} + \rho v \frac{\partial v}{\partial y} = -\frac{\partial p}{\partial y}$$

Dividing the continuity equation by  $\rho_0$  and using the definitions of the new velocity components as given, we get:

$$\frac{\partial}{\partial x} \left( \sqrt{\frac{\rho}{\rho_0}} u^* \right) + \frac{\partial}{\partial y} \left( \sqrt{\frac{\rho}{\rho_0}} v^* \right) = 0$$

$$\therefore \sqrt{\frac{\rho}{\rho_0}} \left( \frac{\partial u^*}{\partial x} + \frac{\partial v^*}{\partial x} \right) + u^* \frac{\partial}{\partial x} \left( \sqrt{\frac{\rho}{\rho_0}} \right) + v^* \frac{\partial}{\partial y} \left( \sqrt{\frac{\rho}{\rho_0}} \right) = 0$$

The last two terms in the last equation represent the steady-state form of the material derivative of the square root of the density ratio. For an incompressible fluid, this quantity will be zero. Then the continuity equation becomes:

$$\frac{\partial u^*}{\partial x} + \frac{\partial v^*}{\partial y} = 0 \quad (1.15)$$

Adding the original form of the continuity equation to each of the components of the momentum equation, and dividing throughout by the constant  $\rho_0$ , yields the following form of the momentum equations:

$$\frac{\partial}{\partial x} \left( \frac{\rho}{\rho_0} u^2 \right) + \frac{\partial}{\partial y} \left( \frac{\rho}{\rho_0} u v \right) = - \frac{1}{\rho_0} \frac{\partial p}{\partial x}$$

$$\frac{\partial}{\partial x} \left( \frac{\rho}{\rho_0} u v \right) + \frac{\partial}{\partial y} \left( \frac{\rho}{\rho_0} v^2 \right) = - \frac{1}{\rho_0} \frac{\partial p}{\partial y}$$

Using the definitions of the new velocity components, these equations become:

$$\frac{\partial}{\partial x} \left( u^{*2} \right) + \frac{\partial}{\partial y} \left( u^* v^* \right) = - \frac{1}{\rho_0} \frac{\partial p}{\partial x}$$

$$\frac{\partial}{\partial x} \left( u^* v^* \right) + \frac{\partial}{\partial y} \left( v^{*2} \right) = - \frac{1}{\rho_0} \frac{\partial p}{\partial y}$$

Expanding the terms on the left side of this equation and using Eq. (1.15) reduces the momentum equations to those of an incompressible fluid. The resulting equations are as follows:

$$\frac{\partial u^*}{\partial x} + \frac{\partial v^*}{\partial y} = 0$$

$$u^* \frac{\partial u^*}{\partial x} + v^* \frac{\partial u^*}{\partial y} = - \frac{1}{\rho_0} \frac{\partial p}{\partial x}$$

$$u^* \frac{\partial v^*}{\partial x} + v^* \frac{\partial v^*}{\partial y} = - \frac{1}{\rho_0} \frac{\partial p}{\partial y}$$

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**FLOW KINEMATICS**



**Problem 2.1**

The following graph was drawn using EXCEL.

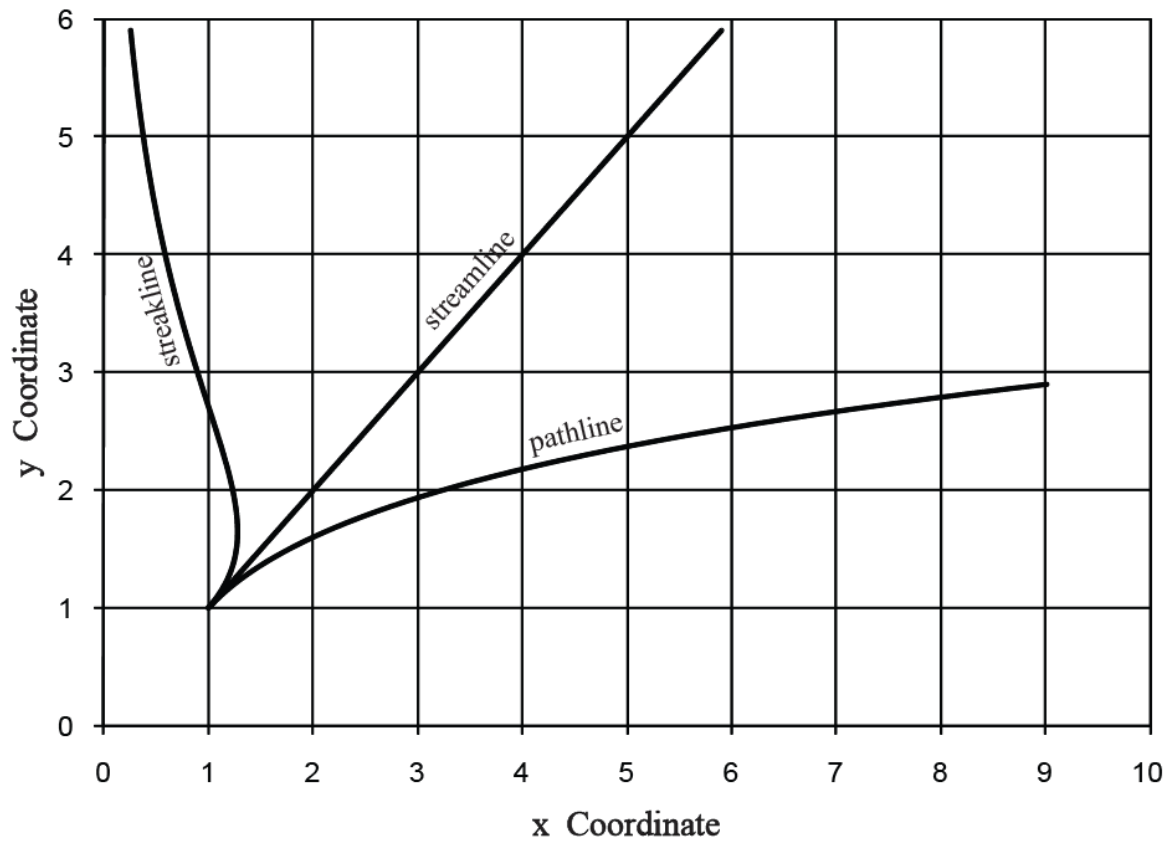


Figure Q.2.1: Streamline, pathline, and streakline for the flow field  $u = x(1+2t)$ ,  $v = w = 0$

**Problem 2.2**

(a) 
$$\frac{dy}{dx} = \frac{v}{u} = 1 + t$$

$$\therefore y = (1 + t)x + C$$

But  $x = y = 1$  when  $t = 0 \Rightarrow C = 0$

Hence at  $t = 0$  the equation of the streamline is:

$y = x$

$$(b) \quad \frac{dx}{dt} = u = \frac{1}{1+t} \quad \frac{dy}{dt} = v = 1$$

$$\therefore x = \log(1+t) + C_1 \quad \therefore y = t + C_2$$

The condition that  $x = y = 1$  when  $t = 0$  requires that  $C_1 = C_2 = 1$ , so that:

$$x = 1 + \log(1+t) \quad y = 1 + t$$

Eliminating  $t$  between these two equations shows that the equation of the pathline is:

$$y = e^{x-1}$$

(c) Here, the equations obtained in (b) above are required to satisfy the condition  $x = y = 1$  when  $t = \tau$ . This leads to the values  $C_1 = 1 - \log(1 + \tau)$  and  $C_2 = 1 - \tau$ . Hence the parametric equations of the streakline are:

$$x = \log(1+t) + 1 - \log(1+\tau) \quad y = t + 1 - \tau$$

At time  $t = 0$  these equations become:

$$x = 1 - \log(1 + \tau) \quad y = 1 - \tau$$

Eliminating the parameter  $\tau$  between these two equations yields the following equation for the streakline at  $t = 0$ :

$$y = 2 - e^{1-x}$$

**Problem 2.3**

$$(a) \quad u = x(1+t) \quad v = 1 \quad w = 0$$

$$\frac{dx}{ds} = x(1+t) \Rightarrow x = C_1 e^{(1+t)s} = C_1 e^s \text{ at } t = 0$$

$$\frac{dy}{ds} = 1 \Rightarrow y = s + C_2$$

But  $x = y = 1$  when  $s = 0$  so that  $C_1 = C_2 = 1$ . Hence:

$$x = e^s \text{ and } y = s + 1$$

$$\therefore x = e^{y-1}$$

(b) 
$$\frac{dx}{dt} = x(1+t) \Rightarrow x = C_1 e^{t(1+t/2)}$$

$$\frac{dy}{dt} = 1 \Rightarrow y = t + C_2$$

But  $x = y = 1$  when  $t = 0$  so that  $C_1 = C_2 = 1$ . Hence:

$$x = e^{t(1+t/2)} \text{ and } y = t + 1$$

$$\therefore x = e^{(y^2-1)/2}$$

(c) As in (b):  $x = C_1 e^{t(1+t/2)}$  and  $y = t + C_2$

But  $x = y = 1$  when  $t = \tau$  so that  $C_1 = e^{-\tau(1+\tau/2)}$  and  $C_2 = 1 - \tau$ . Hence:

$$x = e^{t(1+t/2)-\tau(1+\tau/2)} = e^{-\tau(1+\tau/2)} \text{ for } t = 0$$

and  $y = 1 + t - \tau = 1 - \tau$  for  $t = 0$

$$\therefore x = e^{-(1-y)(3-y)/2}$$

(d) From the continuity equation:

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) + \frac{\partial}{\partial z}(\rho w) = \frac{D\rho}{Dt} + \rho \nabla \cdot \mathbf{u} = 0$$

Hence: 
$$\frac{D\rho}{Dt} = -\rho \nabla \cdot \mathbf{u} = -\rho(1+t) \text{ since } \nabla \cdot \mathbf{u} = (1+t) \text{ here.}$$

i.e. 
$$\frac{D}{Dt}(\log \rho) = -(1+t)$$

Therefore  $\log \rho = -t(1+t/2) + \log C$

So that  $\rho = C e^{-t(1+t/2)}$

But  $\rho = \rho_0$  when  $t = 0$  so that  $C = \rho_0$ . Hence:

$$\rho = \rho_0 e^{-t(1+t/2)} \text{ along the streamline}$$

**Problem 2.4**

The equations that define the streamlines are as follows:

$$\frac{dx_i}{u_i} = ds \quad \text{or} \quad \frac{dx_i}{x_i} = \frac{ds}{(1+t)}$$

$$\therefore \log x_i = \frac{s}{(1+t)} + \log C_i$$

or  $x_i = C_i e^{s/(1+t)}$

Let  $x_i = x_{i0}$  when  $s = 0$ . Then:

$$x_i = x_{i0} e^{s/(1+t)}$$

So that:

$$\boxed{\frac{x}{x_0} = \frac{y}{y_0} = \frac{z}{z_0}} \quad \text{for any time } t.$$

The equations that define the pathlines are as follows:

$$\frac{dx_i}{dt} = u_i \quad \text{or} \quad \frac{dx_i}{dt} = \frac{x_i}{(1+t)}$$

$$\therefore \log x_i = \log(1+t) + \log C_i$$

$$\text{or } x_i = C_i(1+t)$$

Let  $x_i = x_{i0}$  when  $s = 0$ . Then:

$$x_i = x_{i0}(1+t) \quad \text{or} \quad \frac{x_i}{x_{i0}} = (1+t)$$

So that:

$$\boxed{\frac{x}{x_0} = \frac{y}{y_0} = \frac{z}{z_0}} \quad \text{as per the streamlines.}$$

**Problem 2.5**

(a)  $u = 16x^2 + y \quad v = 10 \quad w = yz^2$

On  $0 \leq x \leq 10, y = 0$ :  $\int_0^{10} u dx = \int_0^{10} 16x^2 dx = \frac{16,000}{3}$

On  $0 \leq y \leq 5, x = 10$ :  $\int_0^5 v dy = \int_0^5 10 dy = 50$

On  $10 \geq x \geq 0, y = 5$ :  $\int_{10}^0 u dx = \int_{10}^0 (16x^2 + 5) dx = -\frac{16,000}{3} - 50$

On  $5 \geq y \geq 0, x = 0$ :  $\int_5^0 v dy = \int_5^0 10 dy = -50$

Then, adding the components around the counter-clockwise path gives:

$$\boxed{\Gamma = -50}$$

(b) For the area specified,  $\mathbf{n} = \mathbf{e}_z$  and  $\omega_z = \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) = -1$ .

$$\therefore \int_A \boldsymbol{\omega} \cdot \mathbf{n} dA = \int_0^{10} dx \int_0^5 (-1) dy$$

$$\int_A \boldsymbol{\omega} \cdot \mathbf{n} \, dA = -50$$

This is the same result that was obtained in (a), so that Eq. (2.5) is verified for this flow.

**Problem 2.6**

$$u = \frac{x}{x^2 + y^2} \quad \text{and} \quad v = \frac{y}{x^2 + y^2}$$

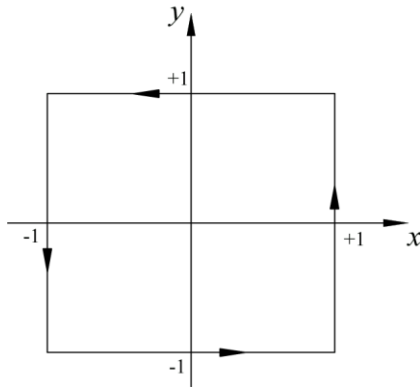
$$\begin{aligned} \therefore \Gamma &= \int_{-1}^{+1} u(x, -1) \, dx + \int_{-1}^{+1} v(+1, y) \, dy + \int_{+1}^{-1} u(x, +1) \, dx + \int_{+1}^{-1} v(-1, y) \, dy \\ &= \int_{-1}^{+1} \frac{-x \, dx}{x^2 + 1} + \int_{-1}^{+1} \frac{y \, dy}{1 + y^2} + \int_{+1}^{-1} \frac{-x \, dx}{x^2 + 1} + \int_{+1}^{-1} \frac{y \, dy}{1 + y^2} \end{aligned}$$

In the foregoing equation, we note that there are two pairs of offsetting integrals. Hence:

$$\Gamma = 0$$

**Problem 2.7**

(a)



$$u = 9x^2 + 2y \quad v = 10x \quad w = -2yz^2$$

$$\int_{-1}^{+1} u(x, -1) \, dx = \int_{-1}^{+1} (9x^2 - 2) \, dx = [3x^3 - 2x]_{-1}^{+1} = 2$$

$$\int_{-1}^{+1} v(+1, y) \, dy = \int_{-1}^{+1} 10 \, dy = [10y]_{-1}^{+1} = 20$$

$$\int_{+1}^{-1} u(x, +1) \, dx = \int_{+1}^{-1} (9x^2 + 2) \, dx = [3x^3 + 2x]_{+1}^{-1} = -10$$

$$\int_{+1}^{-1} v(-1, y) \, dy = \int_{+1}^{-1} (-10) \, dy = [-10y]_{+1}^{-1} = 20$$

Hence:

$$\Gamma = \oint \mathbf{u} \cdot d\mathbf{l} = 2 + 20 - 10 + 20 = 32$$

(b)

$$\boldsymbol{\omega} = \left( \frac{\partial w}{\partial y} - \frac{\partial v}{\partial z} \right) \mathbf{e}_x + \left( \frac{\partial u}{\partial z} - \frac{\partial w}{\partial x} \right) \mathbf{e}_y + \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) \mathbf{e}_z = -2z^2 \mathbf{e}_x + 8 \mathbf{e}_z$$

Hence

$$\boldsymbol{\omega} = -50 \mathbf{e}_x + 8 \mathbf{e}_z$$

on the plane  $z = 5$

(c) For the plane  $z = 5$  the unit normal is  $\mathbf{n} = \mathbf{e}_z$ . Hence:

$$\int_A \boldsymbol{\omega} \cdot \mathbf{n} dA = \int_{-1}^{+1} dx \int_{-1}^{+1} 8 dy = 32$$

This agrees with the result obtained in (a) - as it should, since

$$\Gamma = \oint \mathbf{u} \cdot d\mathbf{l} = \int_A \boldsymbol{\omega} \cdot \mathbf{n} dA$$

**Problem 2.8**

$$u = -\frac{y}{x^2 + y^2} \quad \text{and} \quad v = \frac{x}{x^2 + y^2}$$

(a) 
$$\begin{aligned} \Gamma &= \int_{-1}^{+1} u(x, -1) dx + \int_{-1}^{+1} v(+1, y) dy + \int_{+1}^{-1} u(x, +1) dx + \int_{+1}^{-1} v(-1, y) dy \\ &= \int_{-1}^{+1} \frac{dx}{x^2 + 1} + \int_{-1}^{+1} \frac{dy}{1 + y^2} + \int_{+1}^{-1} \frac{-x dx}{x^2 + 1} + \int_{+1}^{-1} \frac{-dy}{1 + y^2} \\ &= 4 \int_{-1}^{+1} \frac{d\alpha}{1 + \alpha^2} = 4[\tan^{-1} \alpha]_{-1}^{+1} \end{aligned}$$

$$\Gamma = 2\pi$$

(b) 
$$\begin{aligned} \omega &= \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) \\ &= \left[ \frac{1}{(x^2 + y^2)} - \frac{2x^2}{(x^2 + y^2)^2} \right] + \left[ \frac{1}{(x^2 + y^2)} - \frac{2y^2}{(x^2 + y^2)^2} \right] \end{aligned}$$

$$\therefore \omega = 0 \quad \text{provided} \quad x \text{ and } y \neq 0$$

(c) 
$$\frac{\partial u}{\partial x} = \frac{2xy}{(x^2 + y^2)^2} \quad \text{and} \quad \frac{\partial v}{\partial y} = \frac{-2xy}{(x^2 + y^2)^2}$$

$$\therefore \nabla \cdot \mathbf{u} = 0 \quad \text{provided} \quad x \text{ and } y \neq 0$$

**Problem 2.9**

$$u = \alpha y; \quad v = \beta x$$

$$\begin{aligned} \text{(a)} \quad \Gamma &= \oint \mathbf{u} \cdot d\mathbf{l} = \int_{-1}^{+1} (-\alpha) dx + \int_{-1}^{+1} \beta dy + \int_{+1}^{-1} \alpha dx + \int_{+1}^{-1} (-\beta) dy \\ &= [-\alpha x]_{-1}^{+1} + [\beta y]_{-1}^{+1} + [\alpha x]_{+1}^{-1} + [-\beta y]_{+1}^{-1} \\ &= -2\alpha + 2\beta - 2\alpha + 2\beta \end{aligned}$$

$$\Gamma = -4(\alpha - \beta)$$

$$\text{(b)} \quad \int_A \omega \cdot \mathbf{n} dA = \int \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) dx dy = \int (\beta - \alpha) dx dy$$

$$\int_A \omega \cdot \mathbf{n} dA = -4(\alpha - \beta)$$

$$\begin{aligned} \text{(c)} \quad \frac{dx}{ds} &= \alpha y \quad \text{and} \quad \frac{dy}{ds} = \beta x \\ \therefore \frac{dy}{dx} &= \frac{\beta x}{\alpha y} \quad \text{or} \quad \alpha y dy = \beta x dx \\ \alpha \frac{y^2}{2} &= \beta \frac{x^2}{2} - \frac{c^2}{2} \end{aligned}$$

$$\beta x^2 - \alpha y^2 = c^2$$

$$\text{(d)} \quad \alpha = -1 \quad \text{and} \quad \beta = +1 \Rightarrow x^2 + y^2 = c^2 \quad \text{where } u = -y \quad \text{and} \quad v = x.$$

But  $x = 1$  when  $y = 0$  so that  $c^2 = 1$ . Therefore;

$$x^2 + y^2 = 1$$

$$\text{(e)} \quad \alpha = \beta = 1 \Rightarrow x^2 - y^2 = c^2 \quad \text{where } u = y \quad \text{and} \quad v = x.$$

But  $x = 0$  when  $y = 0$  so that  $c = 0$ . Therefore;

$$y = \pm x$$

**Problem 2.10**

The vorticity vector will be in the  $z$  direction and its magnitude will be:

$$\omega(R, \theta) = \frac{1}{R} \frac{\partial}{\partial R} (R u_\theta) - \frac{1}{R} \frac{\partial u_R}{\partial \theta}$$

(a)  $u_R = 0$  and  $u_\theta = \omega R$

$$\frac{\partial}{\partial R} (R u_\theta) = \frac{\partial}{\partial R} (\omega R^2) = 2\omega R$$

and  $\frac{\partial u_R}{\partial \theta} = 0$

$$\therefore \omega(R, \theta) = 2\omega R$$

(b)  $u_R = 0$  and  $u_\theta = \frac{\Gamma}{2\pi R}$

$$\frac{\partial}{\partial R} (R u_\theta) = \frac{\partial}{\partial R} \left( \frac{\Gamma}{2\pi} \right) = 0$$

and  $\frac{\partial u_R}{\partial \theta} = 0$

$$\therefore \omega(R, \theta) = 0 \text{ provided } R \neq 0$$